

REPAIR OF DENTS COMBINED WITH GOUGES CONSIDERING CYCLIC PRESSURE LOADING

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ABSTRACT

Mechanical damage is one of the primary causes of failures in pipe systems. A research program was developed in an effort to quantify the effects of dents with minor scratches on both the burst pressures and fatigue lives of pipelines. Gouges were installed in test specimens at 5% and 10% of the wall thickness, and then dents were installed with a bar placed longitudinally in these notched samples. Dent depths of 5, 10, 15 and 20% of the pipe diameter were studied. One of the primary objectives in this program was to provide pipeline operators with a basic understanding of first, how dent and gouge depths will effect their pipes, and secondly, which defects can be repaired and what is the best procedure for making those repairs. Grinding out the gouge was the restoration method selected for this study. Test specimens were typically tested in pairs so that comparisons could be made between repaired and unrepaired specimens. According to the burst test results, gouge depths of 10% or more combined with dents greater than 15% have burst strengths which are less than 72% SMYS; however, gouges which are less than 5% of the wall in conjunction with dents less than 10% have burst strengths which exceed 100% SMYS. The fatigue tests indicated that dents with gouges definitely act to reduce the fatigue life of pipes, with the level of dent and gouge depth being directly related to their life. In both types of testing, grinding was found to be a suitable method for strength and life restoration.

INTRODUCTION

The project described herein was aimed at demonstrating the feasibility of repairing a shallow gouge in a dent in a pipeline solely by means of grinding out the gouge and associated cracking or other damage. The amount of grinding to be permitted is limited to the extent that the reduced wall thickness after grinding must be adequate to maintain satisfactory pressure carrying capacity. In terms of existing industry practices this can be interpreted to mean 100 percent of SMYS (the specified minimum yield strength of the pipe material). The basis of this project was the hypothesis that any rerounding of a dent that might occur after the removal of the gouge and associated damage would not seriously reduce the burst pressure of the pipe, nor would it significantly affect the fatigue resistance of the pipe.

The method chosen to test the hypothesis involved testing pairs of initially-identical full-scale pipe specimens. One of each pair was to be tested after the gouge had been removed. To compare the effects of the repair on serviceability, some pairs of the specimens were subjected to burst testing while others were subjected to cyclic-pressure fatigue tests. The results of the tests are presented and discussed herein.

BURST TESTING

The primary objective of the burst testing was to determine whether the removal of gouge-damaged material by grinding restores satisfactory serviceability to gouged and intentionally dented pipe. This objective, it was felt, could best be met by bursting pairs of specimens that were damaged identically. In these pairs one would be tested as-damaged and the other would be tested after the gouge-damaged material had been removed by grinding. Since the severity of damage that could be created by means of notches followed by indenting and rerounding with pressure in the pipe was not known, it was necessary to conduct preliminary testing. Thus, a secondary objective was defined in

determining defect parameters that would result in failures of unrepaired specimens with a hoop stress level of at least 60% of the specified minimum yield strength (SMYS). A considerable amount of *preliminary* testing was necessary in order to develop dent/gouge defects which would be realistic and not so severe that their repair would be impractical in field applications. This point can be made more clear when one considers that a dent that is 20% of the pipe diameter will cause a gouge that is 10% of the wall to propagate much deeper than 12.5% of the wall thickness, thus possibly preventing this section from being repaired using conventional grinding techniques. Once the dent/gouge types were adopted based on the *preliminary testing*, fifteen burst tests were conducted to determine the failure pressure for each of the respective defects. The completion of these tests validated the selection of the dents and gouges and a test matrix was developed for the fatigue testing.

Preliminary Work

As discussed previously, the *preliminary* work involved the development of gouges and dents to be used in the fatigue testing. Several steps were involved in this phase of testing, and they are as follows (discussions relating to each will be discussed),

- Selection of pipe based on desired D/t and material grade
- Installation of dents (without gouges) under internal pressure corresponding to 60% SMYS
- Installation of gouges using prescribed geometry and depth relative to pipe wall thickness
- Installation of dents under pressure combined with pre-installed gouges
- Pressurization of all defects after indentation in order to establish the residual dent depths consistent with the typical operating pressure level.

The objective of installing dents without gouges was to determine the initial indentation depth required to obtain residual dent depths that would exist at the operating pressure. The results showed that an initial dent depth of 20% the pipe diameter ($d/D=20\%$) provided a residual dent of 1.58% after the sample had been pressurized to at least 60% of the specified minimum yield strength (SMYS). This observation supported the initial belief that residual dents in thin-walled pipe (wall thickness of 0.188 inches) are unlikely to exceed the 6% value.

The pipe materials selected for this study were parts of two heats of steel. Material testing was done on both heats and the results are provided in **Table 1**. Both materials were 12.75 inch O.D. by 0.188 inch w.t. API 5L Grade X52 line pipe with an ERW longitudinal seam.

Dent Study (Preliminary Work)

Figure 1 is a photograph showing the equipment used in the dent installation such as the hydraulic cylinder, indenter plate, and displacement transducer. During the denting process the hydraulic cylinder forces the 1-inch diameter round bar indenter into the pipe. The indenter used is shown in **Fig. 2**. The dent depth was monitored using a displacement transducer. All dents involved in this research were installed with an internal pressure corresponding to 60% SMYS. The process of denting basically involved the following steps,

- Pipe sample placed in the test rig
- Sample filled with water in preparation for pressurization
- Indenter placed between hydraulic cylinder and pipe and positioned to create dent at the specified location on the pipe (all dents installed 90 degrees from pipe weld seam)
- Pipe sample pressurized to 60% SMYS and maintained while dent installed
- Displacement transducer zeroed and dent installed by increasing pressure to hydraulic cylinder (Displacement and Hydraulic Pressure (Load) both monitored)
- Rerounding permitted by releasing hydraulic cylinder pressure (pressure in pipe permitted to drop, but no water from sample released).
- To effect representative rerounding, sample repressurized to 65% SMYS

The pipe diameters at 0° and 90° relative to the defect were measured at each of the dent locations before denting, after denting (internal pressure removed), and after the sample had been repressurized to 65% SMYS (internal pressure removed). **Table 2** provides the diameters measured in preliminary (no gouge) test specimens before indentation and the diameters measured as a result of the residual dents. Also included in these tables are the loads required to cause the initial dent depths.

As indicated in **Table 2**, the dents are substantially removed from the pipe as a result of pressurization. Previous dent data indicate that residual dent depths greater than 5% are unlikely in pipe with D/t ratios as high as 68 for dents installed with no internal pressure.

Gouge Study (Preliminary Work)

As with the study on dents, the effort in the gouge study was to develop reproducible defects that could be repaired if they were

to actually occur in a pipeline. One of the concerns with the gouge study was the possibility that the penetration of the crack extension at the gouge during rerounding would exceed 12.5% of the wall thickness. With this in mind, specific steps were taken to monitor the propagation of the gouge after indentation. The gouge is created by machining a longitudinal notch into the sample with a radius at the base of 0.002 inches. The tasks listed for the denting process were repeated for the gouge specimens; however, a few additional steps were required in order to quantify the change in gouge depths, and they are as follows,

- Installation of gouges prior to indentation
- Once the dents were installed, the dent/gouge sections were cut circumferentially in order to view the gouges in a cross-sectional manner.
- The gouge in one of the pipe halves was ground in a manner similar to the repair method which would actually be used on a pipeline.

The wall thickness was measured prior to grinding. After grinding dye penetrant was used to check for the remaining gouge. Once the gouge had been successfully removed from the pipe wall, the residual wall thickness was measured. **Figure 3** and **Figure 4** show the gouged region after grinding and after using dye penetrant as a means of detection, respectively. These photographs were taken from the circumferentially-cut pipe segments which were made in the preliminary testing to look at the cross-sectional wall within the defect zone. **Table 3** shows the wall thickness measurements before and after grinding.

As can be seen from **Table 3**, in order to remove all of the gouge, the grinding may require that more than 12.5% of the wall be removed. This is primarily due to the crack extension which is induced during the process of rerounding.

This procedure of cutting the pipe in half allowed the change in gouge depth as a result of indentation to be examined. Based on the results of this exercise, it was found that the maximum gouge/dent combination which could be tested was one that incorporated a 10% gouge with a 20% dent. It should be noted that when the pipe sample was being pressurized to 65% of SMYS, Defect *PG-5* failed. This was a clear indicator that a 20% dent combined with a 15% gouge was too severe for the existing testing conditions. This information, in conjunction with the other test results, was used in developing the test matrix for the actual burst testing.

Test Matrix for Burst Tests

Once all preliminary testing had been completed, a test matrix was developed for the burst tests which would model a range of defects, including the most severe combination from the preliminary testing. **Table 4** shows the test matrix selected for the burst testing. The following nomenclature is used in identifying the specimens,

F2-1N

where: F = Test-type Identifier (*B* for burst and *F* for fatigue)
2 = Material number (in the burst and fatigue tests, this number is a 1 or 2)
1 = Specimen Number
N = Gouge Repair Status (*N* for not ground, *G* for ground, or *D* for dent with no gouge)

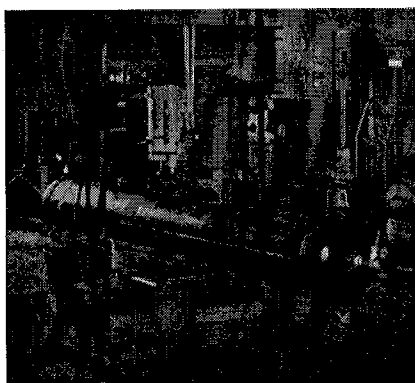


Fig. 1 Pipe Specimen in Dent Installation Rig

All specimens used in the burst (and fatigue) tests utilized 12" nominal size pipe with a wall thickness of 0.188 inches. Although two materials with different heat numbers were obtained for the testing, all fatigue testing involved material from the 2nd lot (refer to **Table 1** for the properties of this material).

Installation of Dents and Gouges

As with the preliminary testing, the gouges were installed prior to indentation. All dents were installed with the pipe pressurized with water to 60% SMYS.

Once all dents had been installed, the specimens were depressurized and the gouges were repaired (i.e. removed by grinding) on those specimens which had been designated to be ground. This was accomplished by means of a hand-held grinder until the gouge was no longer visible using the dye-penetrant detection technique. The grinding was performed with no internal pressure in the pipe. Like the development required for the defect combinations, a similar procedure was required to develop a method for installing dents. Four basic procedures were used and are listed below in addition to those specimens which utilized each procedure. The final procedure to be used was *Procedure B*.

Procedure A (Initial procedure)

Specimens: B1-1N, B1-2G, B1-3N, B1-4G, B1-6N, B1-7N, B1-8N, B1-9G, F2-1N

- | | |
|----------------------------------|--------------------------|
| 1. Machine Notch | 6. Grind if Appropriate |
| 2. Pressurize to 920 psi | 7. Pressurize to 994 psi |
| 3. Indent/Reround/Pressure Decay | 8. Depressurize |
| 4. Depressurize | 9. Measure Dent |
| 5. Measure Dent | |

Procedure B (Improved procedure)

Specimens: B1-10G, B1-11N, B2-12G, B2-13N, B2-14G, B2-15N, F2-2G, F2-3N, F2-4G, F2-5N, F2-6G, F2-7N, F2-8G, F2-9N, F2-10G, F2-11N, F2-12G

- | | |
|----------------------------------|---|
| 1. Machine Notch | 7. Measure Dent |
| 2. Pressurize to 920 psi | 8. Grind if Appropriate |
| 3. Indent/Reround/Pressure Decay | 9. Repressurize to 994 psi |
| 4. Apply Dye Penetrant | 10. Depressurize |
| 5. Repressurize to 920 psi | 11. Measure Dent |
| 6. Depressurize | 12. Pressurize to Failure or Cycles Until Failure |

Procedure C (Specimens with no gouges)

Specimens: PD-5, PD-6, PD-7, PD-8, B1-5D

- | | |
|----------------------------------|--------------------------|
| 1. Pressurize to 920 psi | 5. Pressurize to 994 psi |
| 2. Indent/Reround/Pressure Decay | 6. Depressurize |
| 3. Depressurize | 7. Measure Dent |
| 4. Measure Dent | |

Procedure D (Specimens examined by sectioning)

Specimens: PG-1, PG-2, PG-3, PG-4, PG-5

- | | |
|----------------------------------|---|
| 1. Machine Notch | 7. Depressurize |
| 2. Pressurize to 920 psi | 8. Measure Dent |
| 3. Indent/Reround/Pressure Decay | 9. Cut Circumferentially Through Defect |
| 4. Depressurize | 10. Grind out Crack |
| 5. Measure Dent | 11. Examine Depth of Remaining Crack |
| 6. Pressurize to 994 psi | |

Testing of the samples was conducted by hydrostatically increasing the internal pressure in the specimen until failure occurred. **Figure 5** shows a sample in the test chamber after testing.

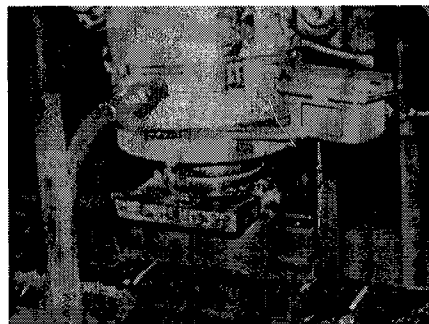


Fig. 2 Close-up View of Denting Set-up



Fig. 3 Grinding of Gouge

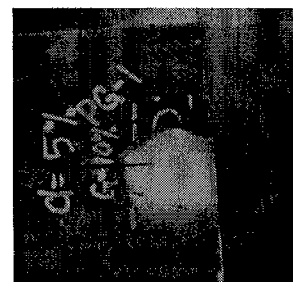


Fig. 4 Dye Penetrant Check



Fig. 5 Failure in Pipe Specimen

Results from Experimental Burst Testing

As with previous testing, the pipe diameters were recorded for each of the specimens before denting, after denting, and after a pressure corresponding to 65% of SMYS had been applied to the specimen. Table 5 provides data relating to these measurements. Also included in this table are the residual dents which remained in the samples as a result of pressurization.

Table 6 records the burst test data for each of the specimens. Recorded in this table is information relating to defect configuration and its respective burst pressure.

Initial testing involved Specimens *B1-1N* through *B1-7N*. From this specimen group a range of defects based on burst pressures was developed. As seen from the above table, Specimen *B1-2G* had the most severe defect in this group with a burst pressure of 625 psi. Specimens *B1-8N* and *B1-9G* were designed to validate the original test envelope. Unfortunately, the dent depth of 12% did not lower the burst pressure as expected; however, the upper bound for the burst pressures was confirmed. Specimens *B1-8N* through *B2-15N* were used to fine tune the test matrix and were specifically targeted at developing defect combinations that would be severe enough for fatigue testing.

Discussion of Burst Test Results

As stated previously, the primary objective of the burst testing was to determine whether the removal of gouge-damaged material by grinding restores satisfactory serviceability to gouged and dented pipe. The range of repairable defects was also an issue. The ideal defect combinations were those which produced failures between 72% and 100% of SMYS. Figure 6 presents a plot of the burst test data in the form of their respective burst pressures as functions of dent and gouge depths.

The subsequent remarks are made based on the results from the burst tests,

- The linear curve fit for the results involving specimens repaired by grinding indicates that when a defect was repaired in this manner, the burst strength of the pipe exceeded 100% SMYS for the defect combinations studied.
- In considering the defects which were not ground, it is clear that the failure pressures decreased with increasing initial dent depth to levels well below 100% SMYS.
- From this one can readily conclude that removal of the gouges by grinding effectively restored the serviceability of the pipe as long as 80% of the nominal wall thickness remains after the removal of the damaged material.

FATIGUE TESTING

While the issue of pressure-cycle fatigue is a greater concern with liquid lines than gas lines, pressure cycling provides a very effective means to quantify the effects that dents with minor scratches have on the lives of pipelines. There were two primary aims in conducting the fatigue tests. The first was to quantify the degree of benefit to be derived from repair by grinding in terms of the fatigue lives of pipes subjected to cyclic internal pressure variations. The second was to show that the lives after grinding would be longer than the useful life of the pipeline.

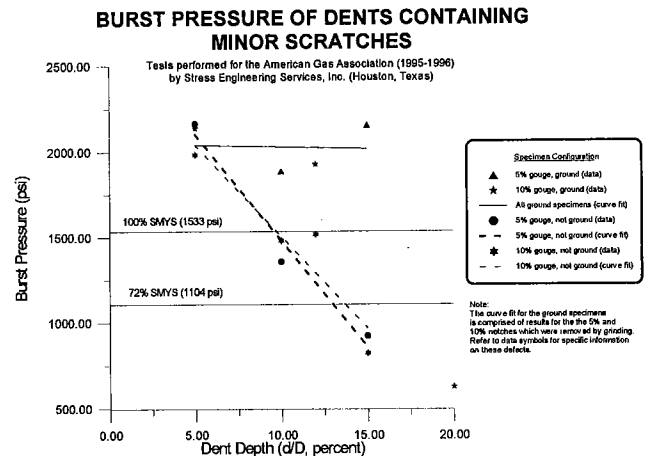


Fig. 6 Burst Pressure of Dents Containing Minor Scratches

Development of the Fatigue Test Matrix

As with the burst tests, a fatigue testing matrix was developed in order to meet the research objectives. Table 7 provides the specimens used in this phase of testing with their respective defective combinations. As before, dent depths listed are before rerounding.

The selection of the defects used in the fatigue testing is based upon results from the burst tests. For example, the high burst pressure for the 5% dents and 5% gouges indicated that the severity of this defect was not sufficient enough to consider its use in fatigue testing.

The fabrication of the samples was identical to the procedure used in the burst testing. All fatigue samples used Procedure B in their indenting process, except sample *F2-1N* which was not tested due to failure during indentation.

Fatigue Testing Experimental Procedures

In conducting the fatigue tests, cyclic internal pressures were applied to the pipes with the pressure range based on a percentage of MAOP. Water was used as the testing medium. Figure 7 shows a schematic diagram of the fatigue testing facility.

The selection of the pressure range was based on previous research which involved samples with reasonable pressure variations, but at the same time had sufficient amplitudes to induce failures within 50,000 cycles of operation. Based on these requirements, the following pressures were applied,

1. 25,000 cycles (or until failure) with $\Delta P = 50\% - 100\%$ MAOP
2. 25,000 cycles (or until failure) with $\Delta P = 0\% - 100\%$ MAOP

This selection of pressures was well-suited for the given defects when it is considered that all samples failed before 50,000 cycles were reached. An additional benefit in selecting pressure variations based on percentages of MAOP is that direct comparison of results from pipe samples with different pipe geometries (D/t) and defect characteristics (gouge and dent depths) can be made. Under normal circumstances, comparison

of the fatigue lives for defects having different applied cyclic pressures is difficult; however, usage of MAOP provides normalization to the results. The presentation of results in Section 3.3 discusses the mathematical method used to determine an equivalent number of cycles for samples cycled with different pressure differentials.

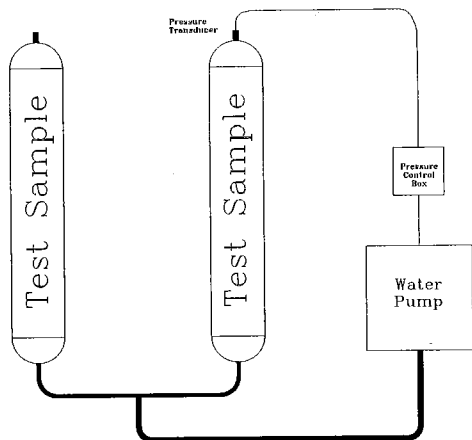


Fig. 7 Schematic Diagram of Fatigue Test Set-up

Results from Experimental Fatigue Testing

All fatigue samples listed in Table 7 were tested until failure, except samples F2-1N and F2-1G which were aborted when the gouged sample failed during the installation process. Although the failures in the burst tests typically involved ruptures (as shown in Fig. 8), most but not all of the fatigue failures resulted in leaks. A typical leak due to fatigue testing is shown in Fig. 9.

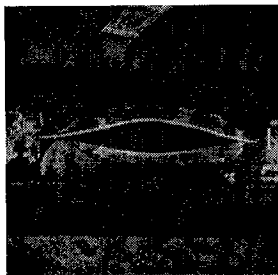


Fig. 8 Photograph of a Typical Failure in a Burst Specimen

Table 8 provides a listing of the fatigue specimens and the number of cycles at which they failed. Because all samples had the same pipe geometries and material properties, the applied pressures were identical for each.

The Equivalent Number of Cycles is used to normalize the data so that the cumulative damage imposed by the multiple pressure cycles (two in these tests) can be incorporated into one value. The Equivalent Number of Cycles is calculated using an equation based on a combination of Miner's Rule and the DOE-B curve. This method calculates an equivalent number of cycles at a specified pressure for a pipe which was pressure cycled at other pressure ratios.

This equation is presented in addition to an example problem.

$$N_{B\ eq} = N_{B_1} \left(\frac{\Delta P}{\Delta P_{B_1}} \right)^{-4} + N_{B_2} \left(\frac{\Delta P}{\Delta P_{B_2}} \right)^{-4}$$

where: $N_{B\ eq}$ = Equivalent number of cycles for Sample B at the specified pressure differential, ΔP
 ΔP = Base pressure differential
 N_{B_1} = Number of cycles obtained for Sample B at ΔP_{B_1}
 ΔP_1 = First pressure differential for Sample B
 N_{B_2} = Number of cycles obtained for Sample B at ΔP_{B_2}
 ΔP_2 = First pressure differential for Sample B

Example Problem

Assume that Sample B had the following fatigue data,
 25,000 cycles at $\Delta P = 500$ psi
 13,000 cycles at $\Delta P = 1200$ psi

Determine the equivalent number of cycles for $\Delta P = 1000$ psi,

$$N_{1000} = 25000 \left(\frac{1000}{500} \right)^{-4} + 13000 \left(\frac{1000}{1200} \right)^{-4} = 28,519 \text{ cycles.}$$

This procedure was done for all data found in Table 9. In addition to the tabulated values, a graphical presentation of these results is presented in Fig. 10. The information in this graph plots gouge depth as a function of cycles to failure for various dent depths.

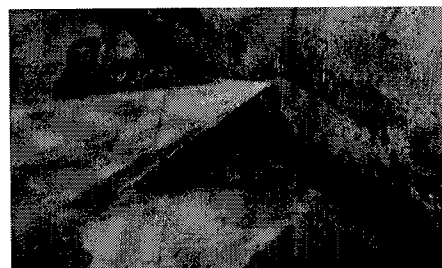


Fig. 9 Photograph of a Typical Failure in a Fatigue Specimen

Discussion of Fatigue Results

From the fatigue tests, the following important observations can be made,

- The contribution of grinding to the fatigue life for pipes cannot be over-emphasized. The fatigue life for ground specimens is approximately five times that of their unground counterparts.
- As would be expected, both increasing dent depth and gouge depth act to reduce fatigue life. Based on the data, it is not apparent which of these contributes most to this reduction. Previous research indicates that by themselves, minor gouges and dents are not severe; however, combinations of the two can be significant. The reason for their severity is the development of microcracks that occur at the base of the gouge when the rerounding occurs.

CONCLUSIONS

One of the primary motivations for conducting this research is to provide pipeline operators with a means for repairing mechanical damage in a simple and effective manner. The results of this study indicate that the repair of gouges in dents by grinding can restore adequate serviceability to a pipeline that has been damaged in a manner which does not involve extremely deep penetration of cracking or associated material damage.

In addition, the burst testing provided several important observations regarding the impact that gouges combined with dents have on the burst strength for a given pipe section.

- Gouges in dents that are repaired by grinding can be expected to have burst strengths that exceed 100% SMYS as long as at least 80% of the nominal wall thickness remains after removal of the damaged material.

- The burst strengths for shallow gouges combined with dents are directly related to the maximum level of indentation and the subsequent rerounding that takes place because of the internal pressure.
- The indentation and rerounding of a pressurized notched pipe as was done herein appears to adequately simulate the effects of real mechanical damage based on the experience of the authors.

The results of the fatigue tests indicate that grinding is an effective means for restoring the pressure-cycle fatigue resistance of a mechanically-damaged pipe.

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FATIGUE LIFE OF DENTS CONTAINING MINOR SCRATCHES

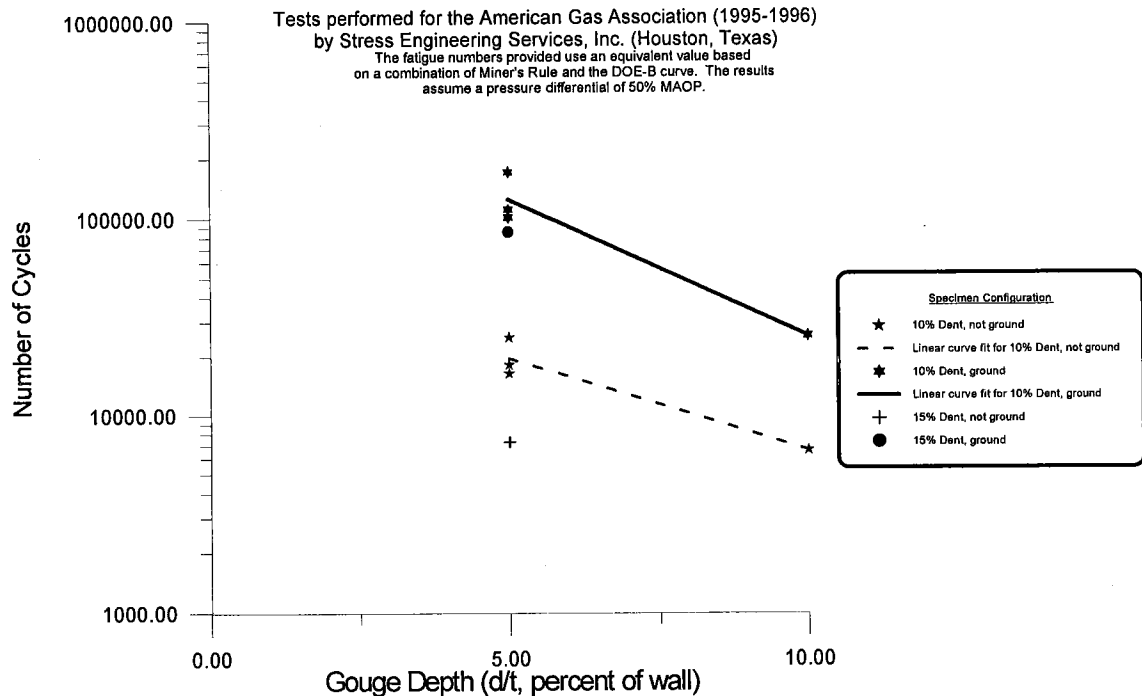


Fig. 10 Fatigue Life as a Function of Gouge Depth

Table 1 Material Properties for Piping Specimens

| Heat # | Yield Strength (psi) | Tensile Strength (psi) | Elongation (%) | Charpy V-notch Impact Value (ft.-lbs.) ¹ |
|--------|----------------------|------------------------|----------------|---|
| F83966 | 53,600 | 72,100 | 34.0 | 17.0 |
| F83967 | 54,300 | 74,100 | 29.5 | 13.0 |

Note:

1. These values determined by means of 1/3-size transverse, flattened specimens which exhibited 100 percent shear area on their fractured surfaces.

Table 2 Measured Dent Depths for Preliminary Testing

| Specimen | Before Denting (inches) | | Diameters due to Residual dents (inches) | | Desired Dent (d/D, %) | Dent Load (lbs.) | Residual Dent (d/D, %) |
|----------|----------------------------|-----------|---|-----------|-----------------------------|------------------------|------------------------------|
| | 0 - 180° | 90 - 270° | 0 - 180° | 90 - 270° | | | |
| PD-5 | 12.728 | 12.842 | 12.668 | 12.826 | 5 | 30,940 | 0.47 |
| PD-6 | 12.685 | 12.877 | 12.587 | 12.848 | 10 | 49,725 | 0.77 |
| PD-7 | 12.650 | 12.902 | 12.558 | 12.871 | 15 | 72,930 | 0.73 |
| PD-8 | 12.712 | 12.826 | 12.511 | 12.917 | 20 | 88,400 | 1.58 |

Table 3 Wall Thickness Changes after Grinding for Preliminary Burst Tests

| Specimen | Gouge Depth (%, d/t) | Wall Thickness Before Grinding (inches) | Wall Thickness After Grinding (inches) | Percentage of Wall Remaining (%) |
|----------|-------------------------|---|--|--|
| PG-1 | 5 | 0.194 | 0.170 | 87.6 |
| PG-2 | 5 | 0.198 | 0.172 | 86.9 |
| PG-3 | 10 | 0.192 | 0.166 | 86.5 |
| PG-4 | 10 | 0.194 | 0.160 | 82.5 |
| PG-5 | 15 | 0.194 | Specimen failed upon rerounding | |

Table 4 Test Matrix for Burst Tests

| Specimen | Gouge Depth (d/t) | Dent Depth (d/D) |
|----------|----------------------|---------------------|
| B1-1N | 5% | 5% |
| B1-2G | 10%, Ground | 20% |
| B1-3N | 10% | 5% |
| B1-4G | 10%, Ground | 5% |
| B1-5D | --- | 5% |
| B1-6N | 10% | 10% |
| B1-7N | 15% | 15% |
| B1-8N | 10% | 12% |
| B1-9G | 10%, Ground | 12% |
| B1-10G | 5%, Ground | 15% |
| B1-11N | 5% | 15% |
| B2-12G | 5%, Ground | 10% |
| B2-13N | 5% | 10% |
| B2-14G | 5%, Ground | 15% |
| B2-15N | 5% | 15% |

Table 5 Geometry Measurements for Burst Test Specimens

| Specimen | Gouge Depth (d/t, %) | Interim Dent ^{Note 1} (d/D, %) | Before Denting (inches) | | Diameters due to Residual dents (inches) | | Residual Dent ^{Note 2} (d/D, %) |
|----------|-------------------------|--|----------------------------|-----------|---|-----------|---|
| | | | 0 - 180° | 90 - 270° | 0 - 180° | 90 - 270° | |
| B1-1N | 5 | 1.48 (5) | 12.796 | 12.779 | 12.664 | 12.802 | 1.03 |
| B1-2G | 10 (Ground) | 4.09 (20) | 12.775 | 12.811 | Failed ^{Note 3} | | --- |
| B1-3N | 10 | 1.71 (5) | 12.785 | 12.784 | 12.656 | 21.810 | 1.01 |
| B1-4G | 10 (Ground) | 2.25 (5) | 12.788 | 12.802 | 12.606 | 12.832 | 1.42 |
| B1-5D | --- | 1.65 (5) | 12.773 | 12.787 | 12.641 | 12.817 | 1.03 |
| B1-6N | 10 | 4.14 (10) | 12.794 | 12.796 | 12.510 | 12.888 | 2.22 |
| B1-7N | 15 | 6.45 (15) | 12.796 | 12.771 | Failed ^{Note 3} | | --- |
| B1-8N | 10 | 3.70 (12) | 12.799 | 12.771 | 12.517 | 12.887 | 2.20 |
| B1-9G | 10 (Ground) | 5.00 (12) | 12.777 | 12.779 | 12.493 | 12.904 | 2.22 |
| B1-10G | 5 (Ground) | 3.46 (15) | 12.786 | 12.790 | 12.623 | 12.884 | 1.27 |
| B1-11N | 5 | --- (15) | 12.800 | 12.780 | Failed ^{Note 3} | | --- |
| B2-12G | 5 (Ground) | 2.09 (10) | 12.767 | 12.760 | 12.656 | 12.784 | 0.87 |
| B2-13N | 5 | 2.01 (10) | 12.770 | 12.780 | 12.612 | 12.766 | 1.24 |
| B2-14G | 5 (Ground) | 1.88 (15) | 12.688 | 12.781 | 12.656 | 12.932 | 0.25 |
| B2-15N | 5 | 1.81 (15) | 12.730 | 12.750 | 12.672 | 12.873 | 0.46 |

Note:

1. The *Interim Dent* value was measured after the dent had been installed in the pipe at 60% SMYS. No pressure was in the pipe at the time of measurement. The value in parentheses was the maximum indentation level at the time of installation.
2. Residual dent measurements made after the specimen had been pressurized to 65% SMYS.
3. These specimens failed in the process of pressurizing the sample to 65% SMYS.

Table 6 Burst Test Information

| Specimen | Gouge Depth (d/t) | Dent Depth (d/D) | P _{burst} (psi) | % SMYS (P _{burst} /SMYS) |
|----------------------|-------------------|------------------|--------------------------|-----------------------------------|
| B1-1N | 5% | 5% | 2,165 | 141 |
| B1-2G | 10%, Ground | 20% | 625 | 41 |
| B1-3N | 10% | 5% | 1,985 | 129 |
| B1-4G ⁽³⁾ | 10%, Ground | 5% | 2,138 | 139 |
| B1-5D | --- | 5% | 2,160 | 141 |
| B1-6N | 10% | 10% | 1,479 | 96 |
| B1-7N | 15% | 15% | 820 | 53 |
| B1-8N | 10% | 12% | 1,517 | 99 |
| B1-9G ⁽³⁾ | 10%, Ground | 12% | 1,928 | 126 |
| B1-10G | 5%, Ground | 15% | 1,820 | 119 |
| B1-11N | 5% | 15% | 775 | 51 |
| B2-12G | 5%, Ground | 10% | 1,887 | 123 |
| B2-13N | 5% | 10% | 1,354 | 88 |
| B2-14G | 5%, Ground | 15% | 2,153 | 140 |
| B2-15N | 5% | 15% | 920 | 60 |

Note:

1. Pipe dimensions: 12.75" O.D. by 0.188" wall
2. SMYS for this pipe calculated to be 1,533 psi (assuming X52)
3. Results for this specimen not entirely valid because grinding was done after only partial rerounding.

Table 7 Specimens Used in Fatigue Testing

| Specimen | Dent Depth, d/D (%) | Gouge Depth, d/t (%) |
|----------|---------------------|----------------------|
| F2-1N | 15 | 10 |
| F2-2G | 15 | 10 |
| F2-3G | 15 | 5 |
| F2-4N | 15 | 5 |
| F2-5G | 10 | 10 |
| F2-6N | 10 | 10 |
| F2-7G | 10 | 5 |
| F2-8N | 10 | 5 |
| F2-9G | 10 | 5 |
| F2-10N | 10 | 5 |
| F2-11G | 10 | 5 |
| F2-12N | 10 | 5 |

Table 8 Fatigue Test Results

| Sample Number | Number of Cycles 50% Differential ($\Delta P = 550 - 1100$ psi) | Number of Cycles 100% Differential ($\Delta P = 100 - 1200$ psi) ⁽²⁾ | Equivalent Number of Cycles with $\Delta P = 50\%$ MAOP ⁽³⁾ |
|----------------------|--|--|--|
| F2-1N ⁽¹⁾ | --- | --- | --- |
| F2-2G ⁽¹⁾ | ---- | --- | --- |
| F2-3G | 25,427 | 3,747 | 85,379 |
| F2-4N | 7,267 | --- | 7,267 |
| F2-5G | 25,427 | --- | 25,427 |
| F2-6N | 6,582 | --- | 6,582 |
| F2-7G | 27,789 | 8,928 | 170,637 |
| F2-8N | 18,093 | --- | 18,093 |
| F2-9G | 24,970 | 5,338 | 110,378 |
| F2-10N | 24,970 | --- | 24,970 |
| F2-11G | 27,479 | 4,594 | 100,983 |
| F2-12N | 16,316 | --- | 16,316 |

Note:

(1) Testing of these two samples was aborted when the ground sample failed during installation.

(2) The minimum reasonable pressure achieved in cycling for the pump system was 100 psi; therefore, the cycle range was 100 - 1,200 psi as opposed to 0 - 1,100 psi.

(3) The Equivalent Number of Cycles is based upon a combination of Miner's Rule and the DOE-B Curve. See explanation in this section of the report for further details.